

VOLUMETRIC EXPANDER VERSUS TURBINE – WHICH IS THE BETTER CHOICE FOR SMALL ORC PLANTS?

3rd International Seminar on ORC Power Systems,

October 12-14, 2015, Brussels, Belgium

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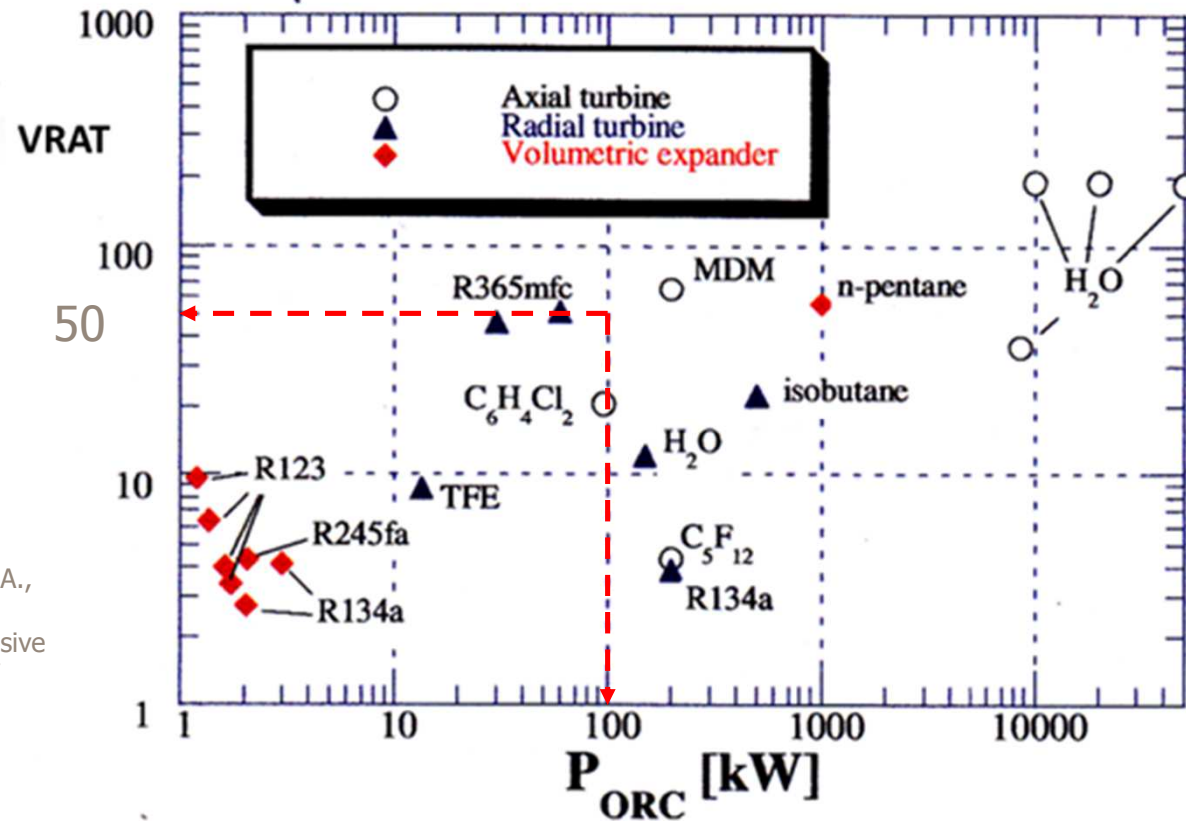
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Motivation

Volume Flow Ratio



(Branchini, L., De Pascale, A., Peretto, A., 2013, Systematic comparison of ORC configurations by means of comprehensive performance indexes, *Applied Thermal Engineering* 61, p 129-140)

Which expander concept is the best compromise, aiming for a „micro expander construction kit“ which enables the designer to implement an appropriate expander for any given application out of a wide range of boundary conditions and working fluids (3- 100 kW_{el})?




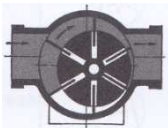
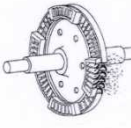


Outline

1. Motivation
- 2. Volumetric Versus Dynamic Expander**
3. Comparison and Assessment of Different Turbines
 - Impulse Versus Reaction Turbines
 - Axial Versus Radial Turbines
4. The Micro Turbo Generator Construction Kit
5. Conclusions

Volumetric Versus Dynamic Expander Selection Criteria for Small ORC Expanders (3-100 kW_{el})

Economic Criteria	Technical Criteria
costs	efficiency
availability on market	rotational speed
reliability	lubrication
maintainability	sealing
	power level
	working fluid
	wear
	complexity
	adaptability

Volumetric Versus Dynamic Expander -Pro & Cons

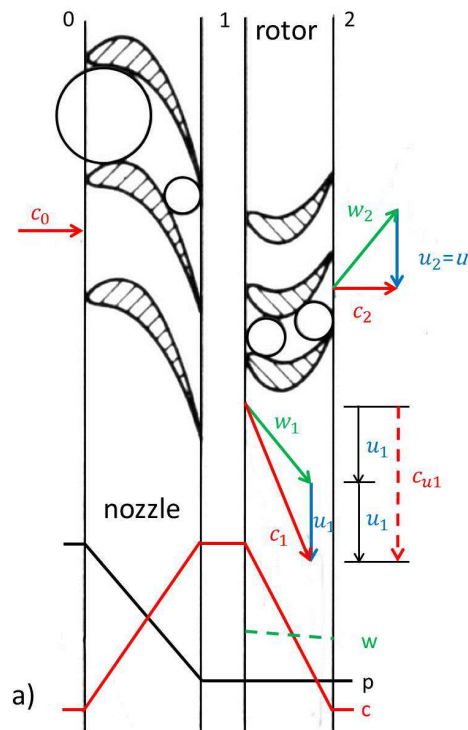
	volumetric expanders				dynamic expanders		
	$work = \int v dp$				$work \sim u^2$		
							
criteria	piston	screw	scroll	vane	axial	cantilever	radial
efficiency	2	1	1	0	1		
VRAT	1	0	0	0	2		
rotational speed	2	1	2	2	0		
part load	2	2	2	2	1		
size	0	0	1	1	2		
adaptability	0	0	2	1	2		
lubrication	0	0	0	0	2		
wear	1	2	1	0	2		
wetness	1	2	2	2	1		
vibration	0	2	2	2	2		
complexity	0	0	2	1	2		
Σ	9	10	15	11	17		

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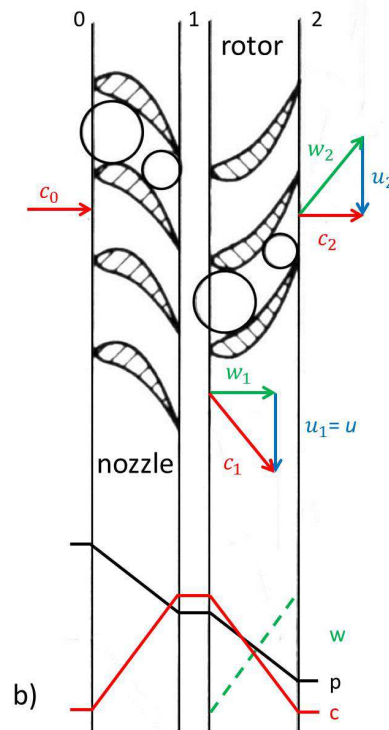
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Comparison and Assessment of Different Turbines

Impulse Versus Reaction Stage



$$\Delta h_{blading, impulse} = 2 * u^2$$



$$\Delta h_{blading, reaction} = 1 * u^2$$

$$\Delta h_{Blading} = c_{u1} * u_1 - c_{u2} * u_2$$


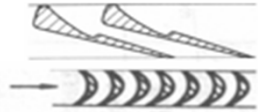

$$u_{opt, impulse} = \frac{1}{\sqrt{2}} * \sqrt{\Delta h_{is}}$$

$$u_{opt, reaction} = 1 * \sqrt{\Delta h_{is}}$$

$$u_{opt, impulse} = u_{opt, reaction} / \sqrt{2}$$

Comparison and Assessment of Different Turbines

Impulse Versus Reaction Turbine


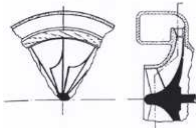

turbine axial – cantilever - radial	impulse	reaction
		
efficiency potential (ts) ≈	80 %	90 %

critierion	impulse	reaction
efficiency	0	2
VRAT	2	0
rotational speed	2	0
axial thrust	2	0
partial admission	2	0
minimal power (size)	2	0
Σ	10	2

Comparison and Assessment of Different Turbines

Axial Versus Radial Turbine

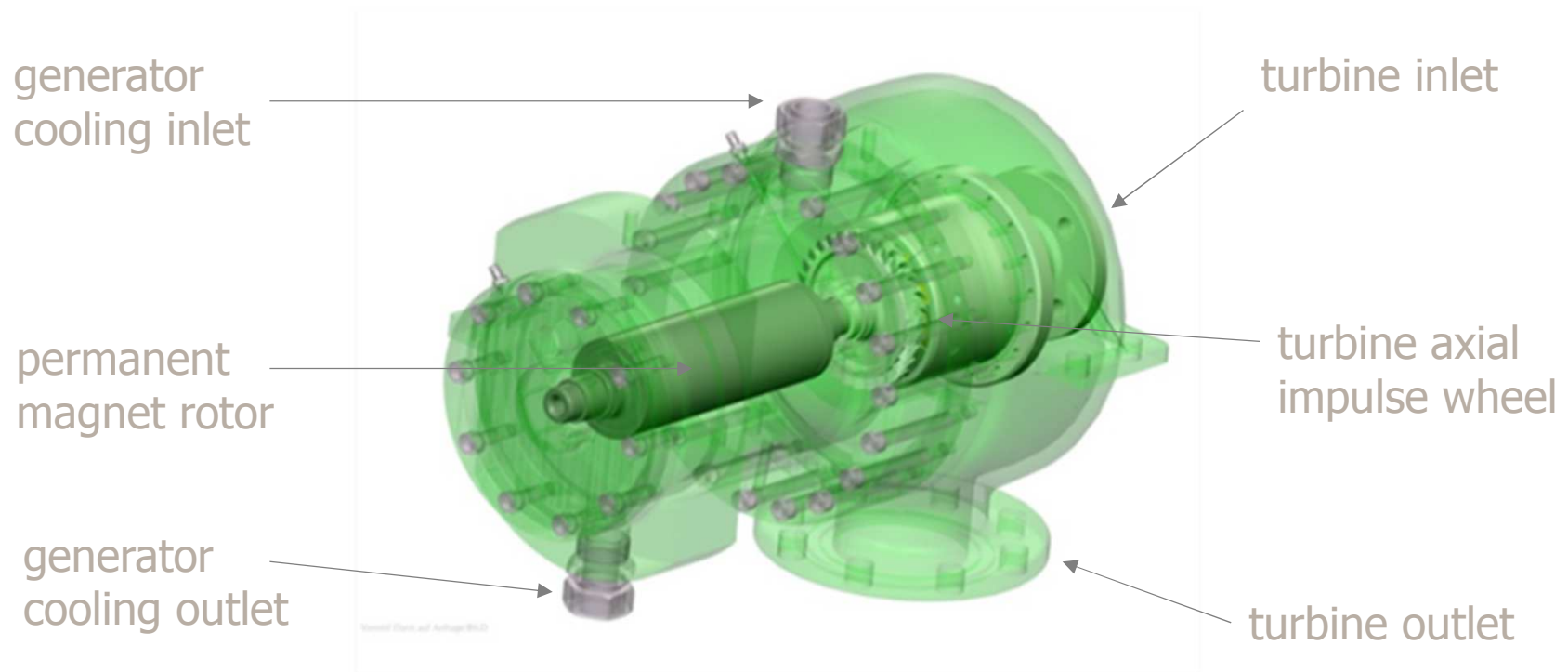
$$\Delta h_{Blading} = c_{u1} * u_1 - c_{u2} * u_2 = \frac{1}{2} * [(c_1^2 - c_2^2) - (w_1^2 - w_2^2) + (u_1^2 - u_2^2)]$$

	turbine type		
			
criterion	axial (r=0)	radial inflow (r ≈ 0,5)	cantilever inflow (r>0)
efficiency	1	2	2
VRAT	2	0	1
rotational speed	2	0	1
axial thrust	2	0	1
partial admission	2	0	2
multi stage	2	0	0
complexity	2	1	2
Σ	13	3	9

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The Micro Turbo Generator Construction Kit (3–100kW_{el})



Several small turbo generators applying small impulse turbines have already been built and successfully tested for various fluids (steam, air, r245fa, SES36, cyclopentane, CO₂ etc.) in the range 1 to 120 kW. Measured efficiencies are in the expected range of 60-70% (is, ts).

Conclusions

- The motivation and goal of this work have been to identify the most suitable expander type for a „micro expander construction kit“ which enables the designer to implement an appropriate expander for any given application out of a wide range of boundary conditions and working fluids (3- 100 kW_{el}).
- The comparison of various volumetric and dynamic expanders types showed, that a turbine is the more flexible concept.
- The evaluation of the pro and cons of impulse versus reaction turbines and an axial versus a radial design led to the conclusion that the simple axial impulse turbine is the best compromise. Therefore, it has been chosen for the „micro expander construction kit“.